

AMME2262 THERMAL ENG 1 NOTES

"thermodynamics, an engineering approach" 7th Ed SI units McGraw-Hill 2011.

"Fundamentals of Heat and Mass Transfer"; 7th edition Wiley. Frank P. Incropera; David P. Dewitt

Contents

Course aims:	7
Introductory concepts:	7
Pressure:	7
Absolute pressure:	7
Gage/vacuum pressure:	7
Pascal's Law:	7
Temperature:	7
Celsius/Kelvin conversion	7
Density:	7
Specific volume:	8
Specific gravity:	8
Specific weight:	8
Properties per unit mole:	8
Thermodynamics system:	8
Closed system:	8
Open system:	8
Continuum assumption applied to system	9
Intensive properties:	9
Extensive property:	9
Specific property	9
Thermal equilibrium:	10
0 th law of thermodynamics:	10
Equation of state	10
Ideal gas equation:	10
List of some gas constants:	11
State postulate:	11
*simple compressible system:	11
Pure substances:	12
Phases of pure substances:	12
Compressed liquid (subcooled liquid):	13

Saturated Liquid:	13
Saturated liquid-vapor mix:	13
Saturated Vapor	14
Superheated vapour:	14
Saturation Temperature and pressure:	15
<i>TSAT</i> :	15
<i>PSAT</i> :	15
Water Saturation curve:	15
Latent heat:	15
Latent heat of fusion:	15
Latent heat of vaporisation	16
Isobaric process:	16
Isothermal process:	16
Phase diagrams	16
Critical point:	16
<i>Pv</i> and <i>Tv</i> Phase diagrams:	17
Example: drawing <i>Pv</i> ; <i>Tv</i> diagrams	17
Three phase diagrams:	19
PT diagram of pure substance:	19
Sublimation:	19
PvT surfaces:	19
Internal Energy:	20
Enthalpy:	20
Entropy:	20
Saturated liquid-vapor mix:	20
Ratio of mass:	20
Average specific volume	20
Volume:	20
Average enthalpy/internal energy	20
Average quantity:	21
Example: Saturated liquid-vapor mix	21
Superheated vapour:	21
Compressed liquids:	21
Compressed liquid property approximations:	22
Tables	22
Interpolation/extrapolation:	22

Using tables examples:	22
Processes and cycles:	23
Process:	23
Path	23
Quasistatic process:	24
Nonquasistatic process	24
Isochoric (or isometric) process:	25
Cycle:	25
Process path: Piston cylinders	29
Compressibility factor:	29
Reduced pressure:	30
Reduced temperature	30
Pseudo Reduced specific volume	30
Nelson-Obert compressibility chart	31
Other equations of state:	32
Van der Walls Equation of state:	32
Beattie-Bridgeman Equation of state:	32
Benedict-Webb-Rubin Equation of state:	32
Peng-Robinson equation of state	32
Generalised compressibility factor:	32
Corrected reduced compressibility factor for H_2, He, Ne, Ar	33
Energy Conversion Efficiencies:	33
Mechanical efficiency:	33
Turbines:	33
Pumps:	33
Efficiency of electrical devices:	34
Connecting efficiencies:	34
Forms of energy:	34
Macroscopic:	34
Internal Energy:	34
Microscopic:	35
Kinetic energy, KE:	35
Potential energy, PE:	35
Total system energy:	35
Power:	35
Energy transferred by heat:	35

Adiabatic process:	35
Heat Transfer mechanisms:	36
Conduction:	36
Convection:	36
Radiation	36
Energy transferred by work:	36
Work:	36
First law of thermodynamics:	38
Watts:	38
Thought experiment:	38
First Law of thermodynamics:	39
Adiabatic systems:	39
Back to thought experiment with first law:	40
Steps for problem solving:	40
Boundary Work:	41
Moving boundary work/ PdV work	41
Polytropic processes:	44
Sign convention of first Law:	45
Energy balance for constant pressure process	46
Expansion in a vacuum:	46
Specific Heat:	48
CV : Specific heat at constant volume	48
CP : Specific heat at constant pressure:	49
Internal energy, enthalpy and specific heats of solids and liquids:	51
Conservation of mass:	52
Closed system:	52
Control volume:	52
Conservation of mass principle:	52
Flow work/ flow energy:	53
Theta term:	54
Energy analysis of steady flow systems:	54
Analysis of unsteady uniform flow process:	58
2^{nd} law of thermodynamics:	60
Processes and the first/second law:	60
Implications of 2^{nd} law:	60
Thermal systems:	60

Thermal energy reservoirs:	60
Heat engine:.....	60
Steam power plant:.....	61
Thermal efficiency:.....	61
Kelvin-Planck statement of second law:	62
Refrigerators:	62
Coefficient of performance:.....	62
Refrigerator.....	62
Heat pump:	62
Clausius statement of second law:	63
Interpretation of cлаusius statement:	63
Equivalence of kelvin/planck and Claudio statement:	64
Reversible processes:.....	65
Irreversibilities:	65
Reversible Processes as idealisation of irreversible processes:.....	65
The Thermodynamic Temperature Scale.....	68
Carnot heat engine is most efficient heat engine:.....	68
Quality of energy:.....	69
Carnot refrigeration:	69
Clausius inequality:	70
Entropy:.....	71
Entropy graphically:	74
3 rd law of thermodynamics:	75
Entropy and disorder:	76
Boltzmann relation:	76
1 st gibbs equation:.....	76
Entropy change in solids and liquids:.....	77
Entropy change in ideal gases:.....	77
Isentropic efficiencies of steady-flow turbines.....	78
Isentropic efficiencies for pumps and compressors	80
effciency of nozzels	80
Steady compressor work for ideal gases with constant specific heats:	81
Steady isentropic compression requires more work than isothermal compression.....	82
Multistage compression with intercooling reduces the required compressor work	82
Reversible carnot cycle:	84
Refrigeration cycle:	84

Idealised vapour-compression refrigeration cycle:	85
The actual vapour-compression cycle (AVCC)	88
The forward carnot engine cycle	90
Air standard assumptions:	90
Air standard cycle:	91
Cold air standard assumptions	91
Overview of reciprocating engine	91
2 and 4 stroke engines:	92
Diesel cycle	96
Dual cycle: an improved CI model	98
Otto engine example analysis:	99
Brayton cycle- ideal cycle for gas turbine engines.	102
Equations:	103
Pressure ratio:	103
efficiency	104
Back work ratio	104
Cold air standard	105
Optimal pressure ratio is:	105
Ideal jet propulsion cycle:	106
Deviation from ideal to actual gas turbine	106
Ideal Rankine cycle	109
Equations:	109
Pump ($q=0$)	109
Boiler ($w=0$)	110
Turbine $q = 0$	110
Condenser ($w=0$)	110
Whole cycle analysis:	110
Efficiency:	110
Actual rankine cycle:	110
Rankine cycle example:	111
Heat transfer	112
Steady conduction and fourier's law:	112
Heat flux	112
Thermal conductivity is large for solids and varies with phase and temperature	113
Convection and newton's law of cooling:	115
Convection coefficient h	116

3 modes of convection: natural, forced and boiling/condensation:	116
Radiation:	117
Stefan-boltzman law:	118

Assessments:

10% tute assignments (8/12)

30% 3 quizzes (5,8,11 on Wednesday)

10% labs

50% final

Course aims:

- Fundamental thermodynamics applied to open and closed systems
- Understanding basic thermodynamic properties of substances
- Ability to analyse in term of 1st and 2nd law of thermodynamics
- Calculate gas and multiphase power and refrigeration cycles

Introductory concepts:

Pressure:

$$P = \frac{F}{A} \quad (\text{N/m}^2 ; \text{Pa})$$

Absolute pressure:

- Measured relative to absolute zero pressure

Gage/vacuum pressure:

- Measured relative to atmospheric pressure

$$P_{abs} = P_{guage} + P_{atm}$$

Pascal's Law:

$$P_1 = P_2$$

Temperature:

- Kelvin (K)

Celsius/Kelvin conversion

$$T(K) = T(^{\circ}\text{C}) + 273.15$$

Density:

$$\rho = \frac{m}{V}$$

Specific volume:

$$v = \frac{V}{m} = \frac{1}{\rho}$$

Specific gravity:

$$SG = \frac{\rho}{\rho_{H_2O}}$$

Specific weight:

$$\rho g$$

Properties per unit mole:

Are denoted with an overbar:

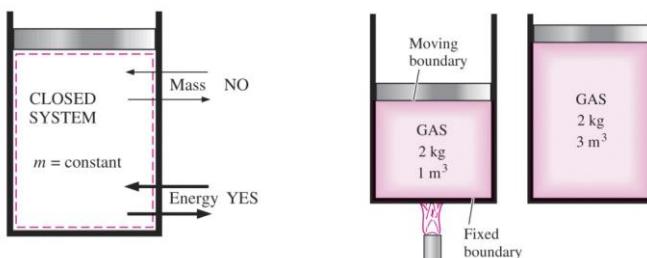
$$\bar{v} = \frac{m^3}{kmol}$$

$$\bar{u} = \frac{kJ}{kmol}$$

$$\bar{h} = \frac{kJ}{kmol}$$

Thermodynamics system:

Could be either **closed** or **open**



Closed system:

Fixed amount of mass, and has no mass flow across boundary. (can have a moving system boundary though).

Referred to as **control mass**.

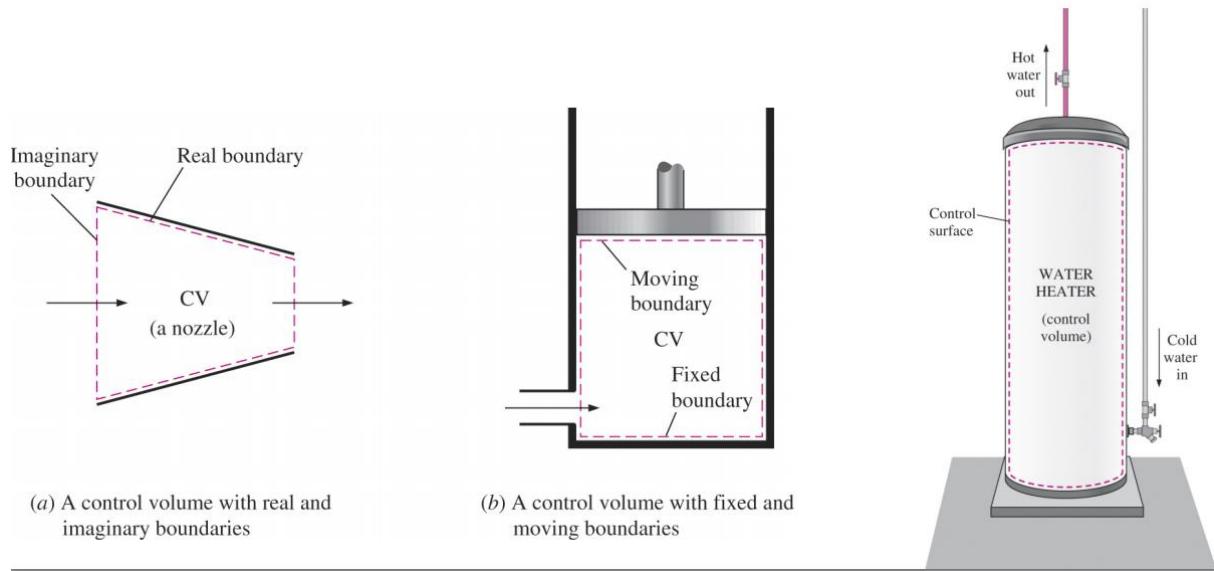
Eg: closed tank or piston cylinder.

Open system:

Involves mass flow across boundary.

Referred to as **control volume**.

Eg: turbine, nozzle, compressor



Continuum assumption applied to system

Matter is made up of atoms that are widely spaced in the gas phase. Yet it is very convenient to disregard the atomic nature of a substance and view it as a continuous, homogeneous matter with no holes, that is, a continuum. Despite the large gaps between molecules, a substance can be treated as a continuum because of the very large number of molecules even in an extremely small volume.

Intensive properties:

Independent of size of system:

- Temperature
- Pressure
- Density

Extensive property:

Dependent on size of system:

- Mass
- Volume
- Energy

Specific property

Extensive properties per unit mass are **specific properties**, and are intensive.

- Mass
- Specific volume: $\frac{V}{m} = \nu$
- Specific internal energy: $\frac{U}{m} = u$
- Specific energy: $\frac{E}{m} = e$

Thermal equilibrium:

A system is in thermodynamic equilibrium if it is in thermal, mechanical, phase and chemical equilibrium. When a system in an equilibrium state the properties of a system do not change and are constant over the entire system.

0th law of thermodynamics:

The Zeroth law of thermodynamics states that if two bodies are in thermal equilibrium with a third body, they also are in thermal equilibrium with each other. This validates temperature measurements in that if two bodies have the same temperature, they are in thermal equilibrium. (Fowler 1931) Two bodies reaching thermal equilibrium after being brought into contact in an isolated enclosure. By replacing the third body with a thermometer, the zeroth law can be restated as two bodies are in thermal equilibrium if both have the same temperature reading even if they are not in contact.

Equation of state

An equation of state is any equation that relates pressure, temperature and specific volume of a substance.

Ideal gas equation:

Ideal gas assumptions:

- The gas is composed of a large amount of small molecules.
- The gas molecules are elastic
- The size and total volume of the molecules is small relative to the volume
- Thermal motions of the gas are random
- ->real gases can be approximated by an ideal gas at “low” densities: “low” pressures and “high” temperatures.

$$PV = nR_u T$$

$$P = \text{Pressure}; V = \text{Volume}; n = \text{mols}; R_u = 8.314 \left(\frac{\text{J}}{\text{K} \cdot \text{mol}} \right); T = \text{temperature (K)}$$

BUT- better to use mass than mols:

So:

$$PV = mRT$$

Cannot use ideal gas equation for steam or refrigerant

$$P = (\text{kPa}); V = (\text{m}^3); m = (\text{kg}); R = \text{gas constant} = \frac{R_u}{\text{molecular weight}} \left(\frac{\text{kJ}}{\text{K} \cdot \text{kg}} \right); T = (\text{K})$$

List of some gas constants:

Gas	J/(kgK)	kJ/(kgK)
Argon, Ar	208	0.208
Acetylene	319	0.319
Ammonia	488	0.488
Carbon Dioxide, CO ₂	188.9	0.1889
Carbon Monoxide, CO	297	0.297
Carbonic acid	189	0.189
Helium, He	2077	2.077
Hydrogen, H ₂	4124	4.124
Methane - natural gas, CH ₄	518.3	0.5183
Nitrogen, N ₂	296.8	0.2968
Oxygen, O ₂	259.8	0.2598
Propane, C ₃ H ₈	189	0.189
Sulfur dioxide, SO ₂ (sulfuric acid)	130	0.13
Air	286.9	0.2869
Water vapor	461.5	0.4615

State postulate:

The state of a simple* compressible system is completely specified by two independent, intensive properties. (i.e- hold one constant and change another property)

*simple compressible system:

No:

- Electrical
- Magnetic
- Gravitational
- Motion

- Surface tension

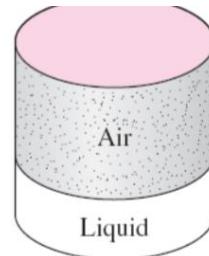
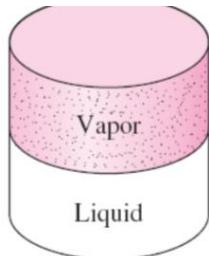
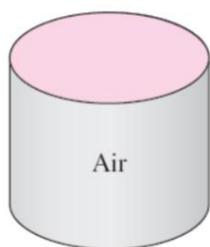
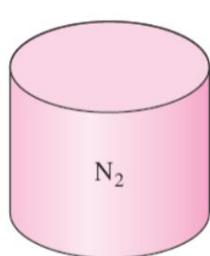
Eg: piston

If $T = 25^\circ C$; $\nu = 0.9 \frac{m^3}{kg}$: This is state postulate

Pure substances:

Have a fixed chemical composition throughout:

- Air is a pure substance at atmospheric pressure and temperature (fixed mixture throughout)

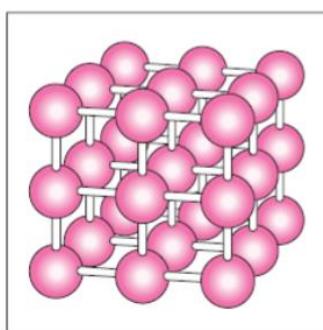


Nitrogen and gaseous air are pure substances.

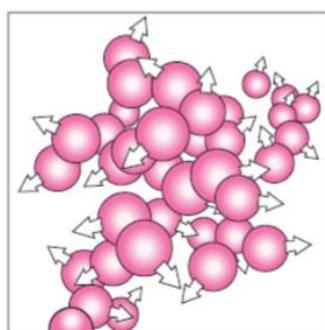
A mixture of liquid and gaseous water is a pure substance, but a mixture of liquid and gaseous air is not.

Phases of pure substances:

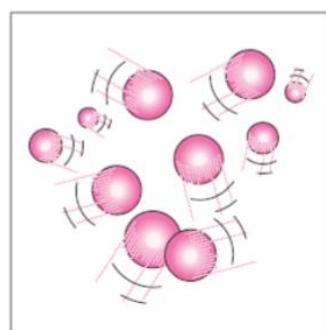
← Order ←
→ Energy →



Solid

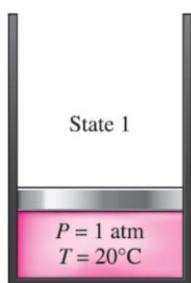


Liquid



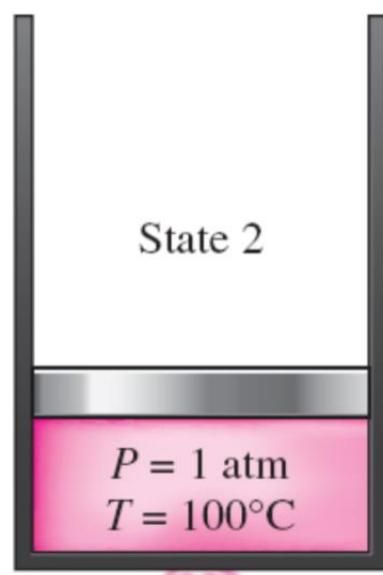
Gas

Compressed liquid (subcooled liquid):
A substance that is **not** about to vaporize



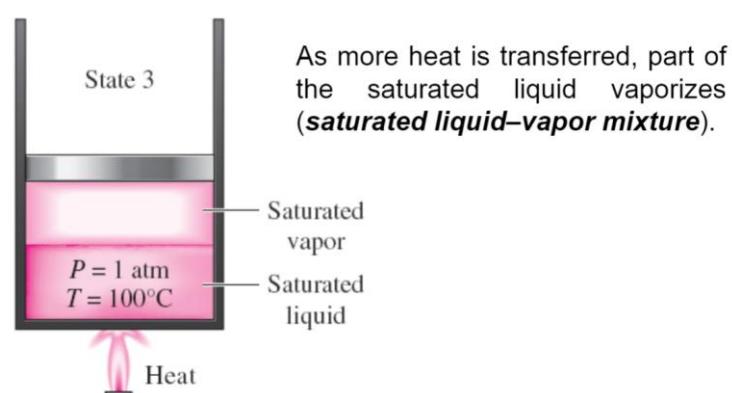
At 1 atm and 20°C , water exists in the liquid phase (**compressed liquid**).

Saturated Liquid:
Liquid is about to vaporize



At 1 atm pressure and 100°C , water exists as a liquid that is ready to vaporize (**saturated liquid**).

Saturated liquid-vapor mix:
State at which liquid and vapour coexist in equilibrium (but temperature is not rising, as energy is used to vaporize liquid)



As more heat is transferred, part of the saturated liquid vaporizes (**saturated liquid-vapor mixture**).

Saturated Vapor

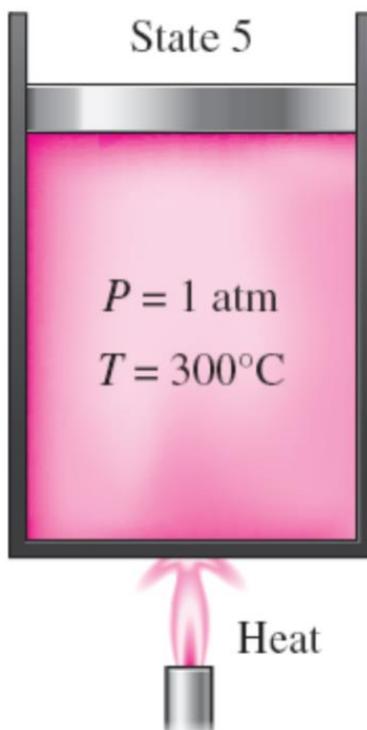
Vapour is about to condense:



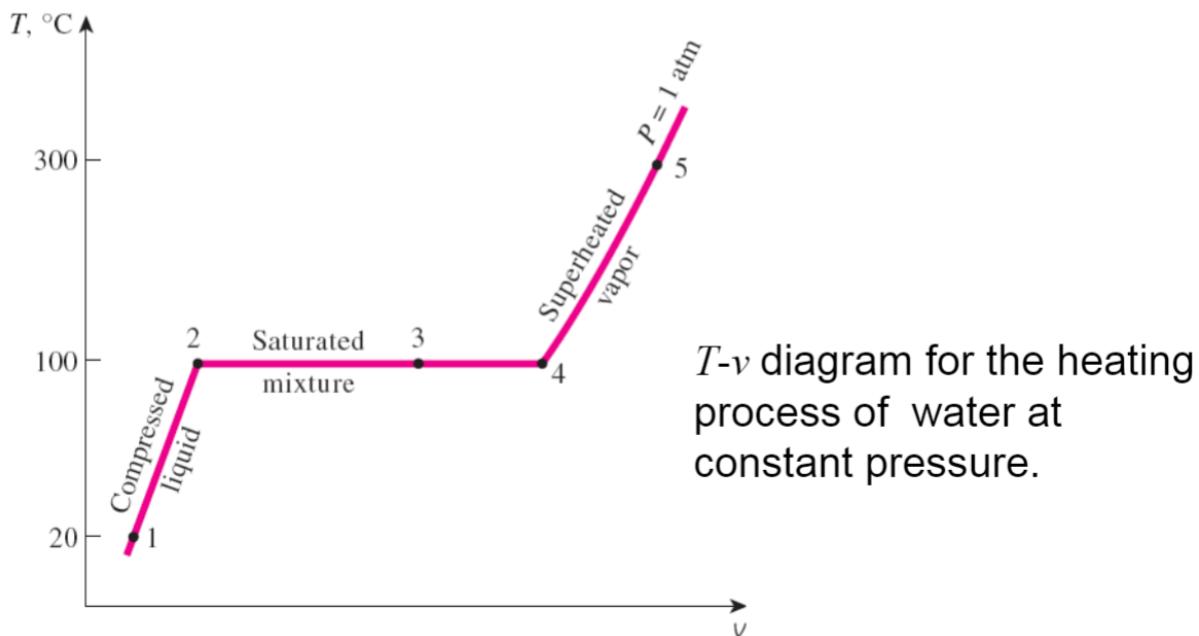
At 1 atm pressure, the temperature remains constant at 100°C until the last drop of liquid is vaporized (**saturated vapor**).

Superheated vapour:

- Too much energy (temperature increases); vapour is NOT about to condense



As more heat is transferred, the temperature of the vapor starts to rise (**superheated vapor**).



Reversing the process following the same path releases the same amount of heat as was added in the forward process.

Saturation Temperature and pressure:

The temperature at which water starts boiling depends on the pressure, so if the pressure is fixed, so is the boiling temperature

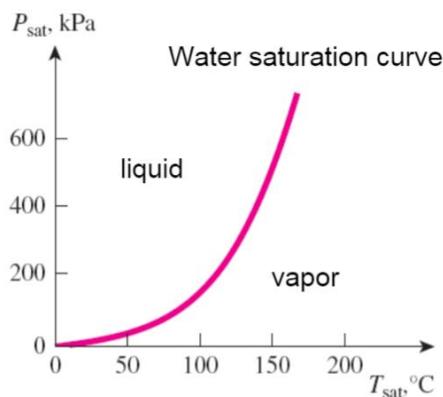
T_{SAT} :

The temperature at which a pure substance changes phase at a given pressure

P_{SAT} :

The Pressure at which a pure substance changes phase at given temperature

Water Saturation curve:



Latent heat:

The amount of energy absorbed or released during a phase change process

Latent heat of fusion:

The amount of energy absorbed during melting (equivalent to amount released in freezing)

Latent heat of vaporisation

The amount of energy absorbed during vaporisation (equivalent to released in condensation)

- Depends on temperature and pressure at which the phase occurs

At 1 atm:

Latent heat of fusion of water = 333.7 kJ/kg

Latent heat of vaporisation = 2265.5 kJ/kg

Isobaric process:

- Constant pressure

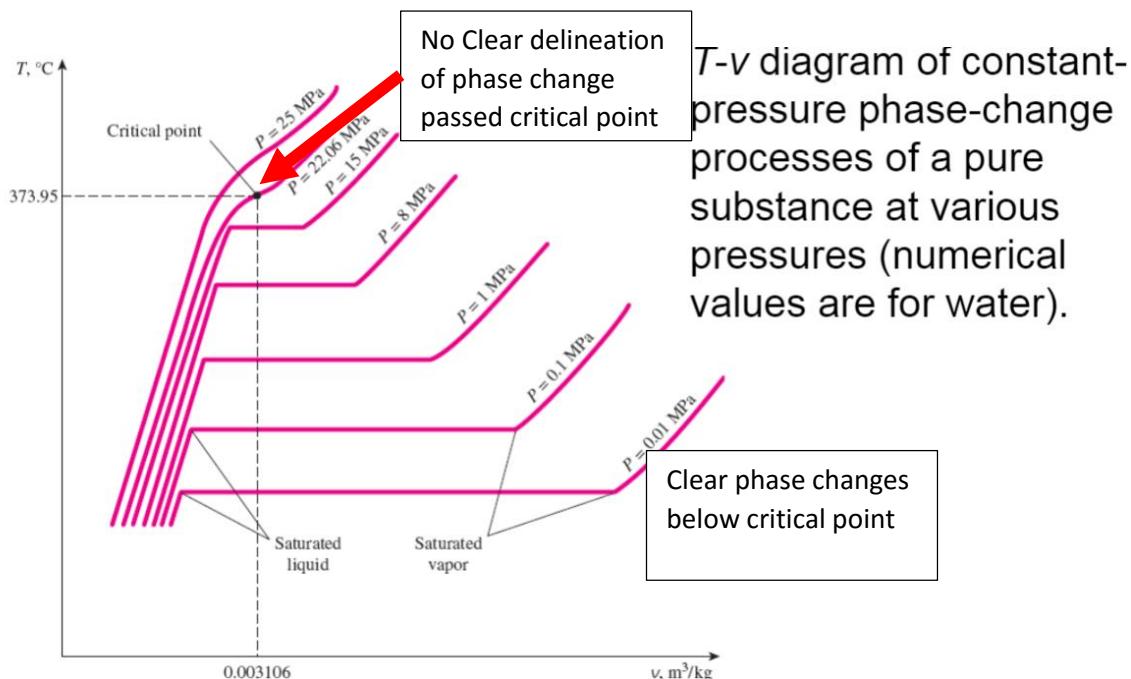
Isothermal process:

Constant temperature:

Phase diagrams

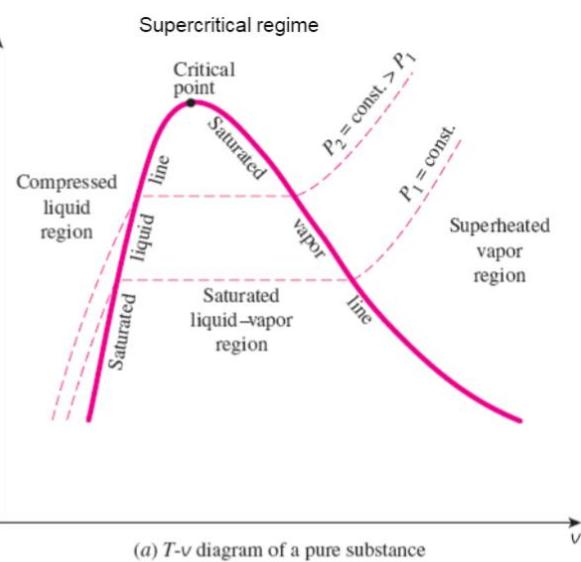
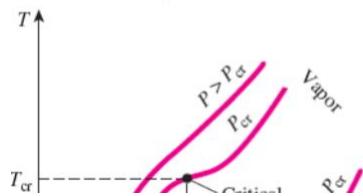
Critical point:

The point at which the saturated liquid and saturated vapour states are identical.

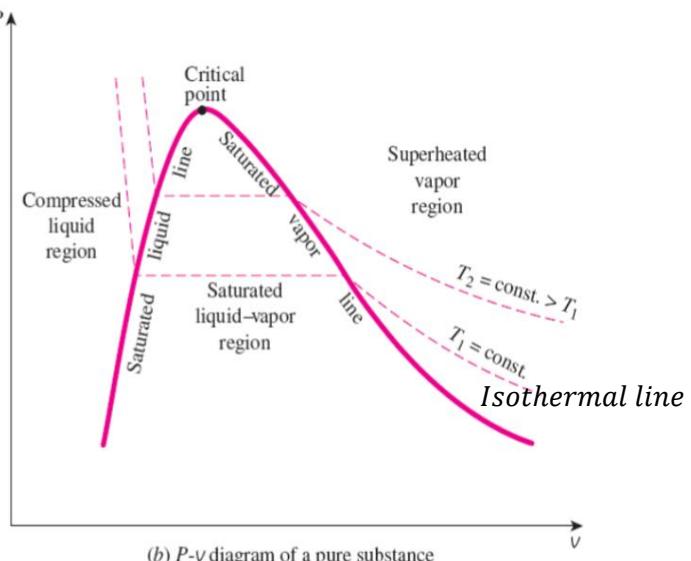
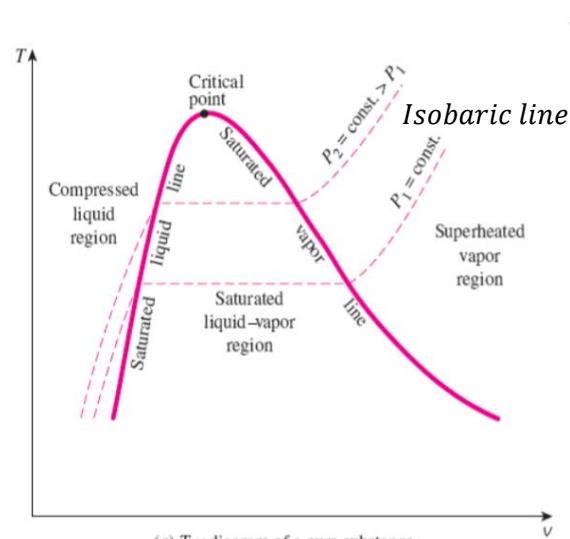


Phase diagrams

- **saturated liquid line**
- **saturated vapor line**
- **compressed liquid region**
- **superheated vapor region**
- **saturated liquid-vapor mixture region (wet region)**
- **Supercritical region**



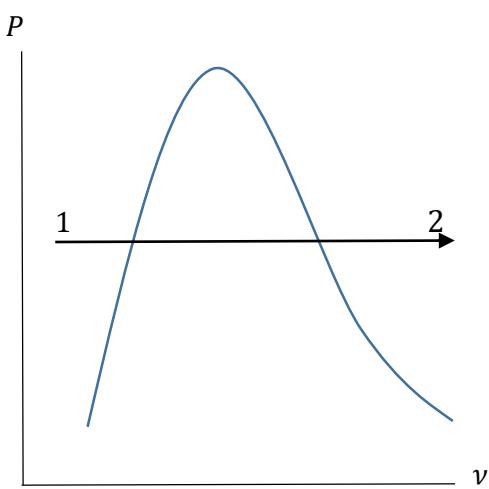
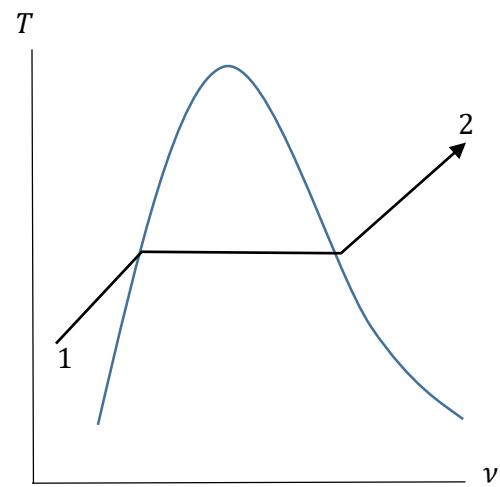
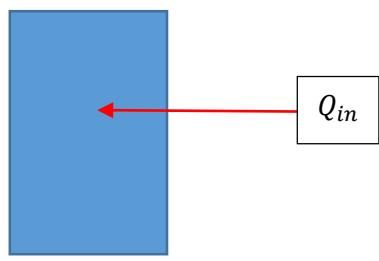
Pv and T_v Phase diagrams:



Notes: line left of critical is saturated liquid line; right is saturated vapour line

Example: drawing *Pv*; *T_v* diagrams

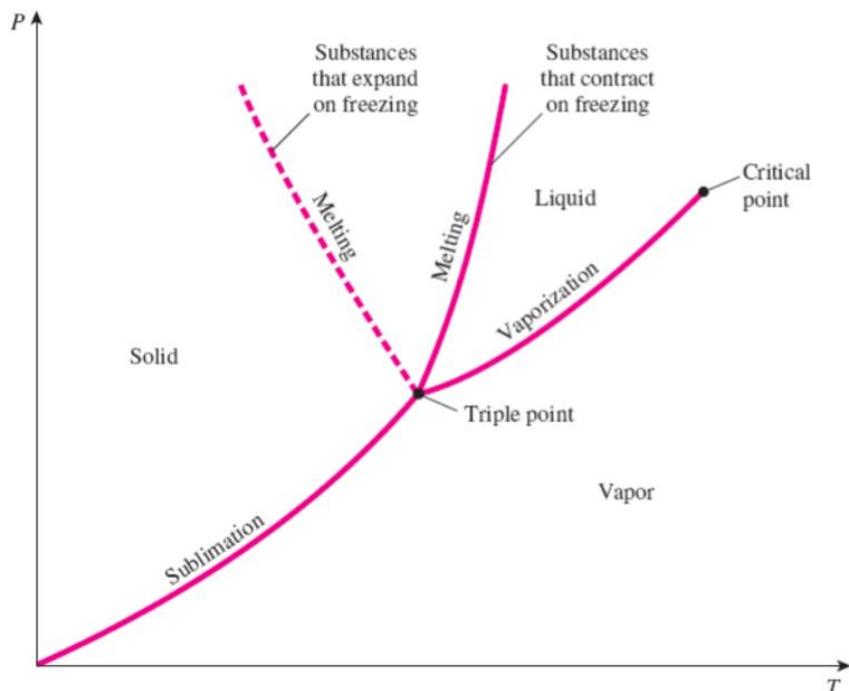
Constant pressure process; water; heat is added to water from a compressed liquid till the water is a superheated vapour. Show on $T - v$; $P - v$ diagrams. (Heat added, pressure constant)



Three phase diagrams:

PT diagram of pure substance:

P-T diagram of pure substances.



Triple point:

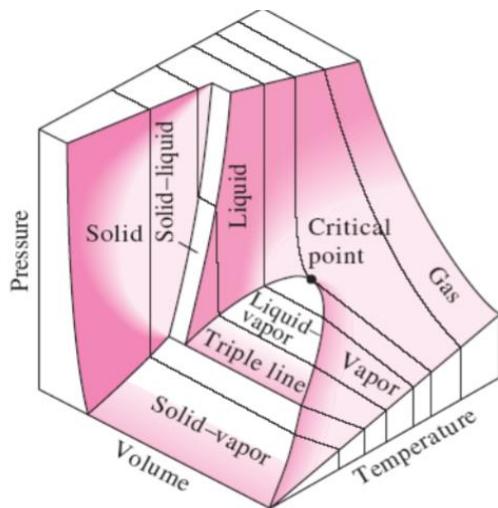
Critical point for 3 states (solid/liquid/gas coexist at same time)

Sublimation:

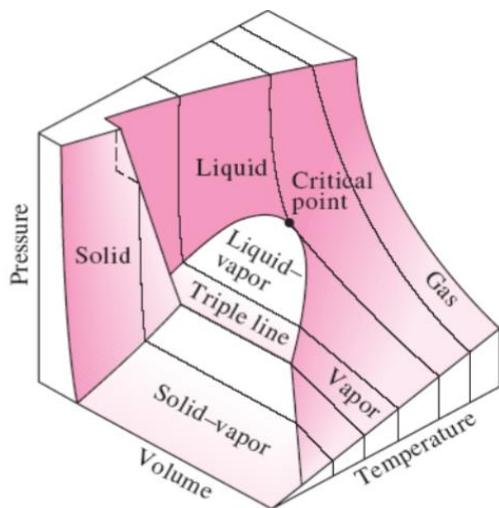
Sublimation: Passing from the solid phase directly into the vapor phase.

PvT surfaces:

Too much info on one diagram:



P-v-T surface of a substance that contracts on freezing.



P-v-T surface of a substance that expands on freezing (like water).

Internal Energy:

Internal energy is the sum of all microscopic forms of energy related to molecular structure and activity (nuclear, thermal, electrical etc)

Can be either Intensive:

$$u: \left(\frac{kJ}{kg}\right)$$

Extensive:

$$U \text{ (kJ)}$$

Enthalpy:

Enthalpy is internal energy + Pressure energy:

Intensive:

$$h = u + Pv \text{ Units: kJ/kg}$$

Extensive:

$$H = U + PV \text{ kJ}$$

Entropy:

- 2nd law of thermodynamics; associated as a measure of disorder

Intensive:

$$s: \text{ kJ/kg} \cdot \text{K}$$

Extensive:

$$S: \text{ kJ/K}$$

Saturated liquid-vapor mix:

We are given either pressure or temperature

We are given one of the following: v, u, h or s

Ratio of mass:

$$x = \frac{\text{mass of vapor}}{\text{total mass}} = \frac{m_g}{m_{total}}$$

$$\therefore m_t = m_f + m_g$$

Average specific volume

$$v_{av} = v_f + x v_{fg}$$

Volume:

$$V = v_{av} m_t$$

Average enthalpy/internal energy

$$u_{av} = u_f + x u_{fg}$$

$$h_{av} = h_f + xh_{fg}$$

Average quantity:

This works for any physical quantity:

$$y = y_{av} = y_f + xy_{fg} \quad (\text{where } y_{fg} = y_g - y_f)$$

Example: Saturated liquid-vapor mix

A rigid container contains 2 kg of vapor and 8 kg of liquid water at 90°C. Determine the volume of the container

Given: $T = 90^\circ\text{C}$

$$m_g = 2\text{kg}$$

$$m_f = 8\text{kg}$$

Assume a saturated mixture:

Determine $V_{\text{container}}$

$$V = v_{av} \times m_{\text{total}}$$

$$m_t = m_f + m_g = 10\text{kg}$$

$$v_{av} = v_f + xv_{fg} = v_f + \frac{m_g}{m_t}(v_g - v_f)$$

Saturated water temperature table A4:

$$v_f = 0.001836 \text{ m}^3/\text{kg}$$

$$v_g = 2.3593 \text{ m}^3/\text{kg}$$

$$\therefore v_{av} = 0.47269 \rightarrow V = 4.73 \text{ m}^3$$

Superheated vapour:

Compared to saturated vapor, superheated vapor is characterized by:

- Lower pressure ($P < P_{\text{sat}}$)
- Higher temperature ($T > T_{\text{sat}}$)
- Higher specific volume ($v > v_g$)
- Higher internal energy ($u > u_g$)
- Higher enthalpy ($h > h_g$)

Compressed liquids:

Compared to saturated vapor, superheated vapor is characterized by

- higher pressure ($P > P_{\text{sat}}$)
- lower temperature ($T < T_{\text{sat}}$)
- lower specific volume ($v < v_g$)

- lower internal energy ($u < u_g$)
- lower enthalpy ($h < h_g$)

The properties of compressed liquids vary strongly with temperature and in many cases negligibly with pressure

Compressed liquid property approximations:

$$y \approx y_{f@T}$$

So: the property $y = v, u, h$ can be approximated by using a saturated liquid property at same temperature

Tables

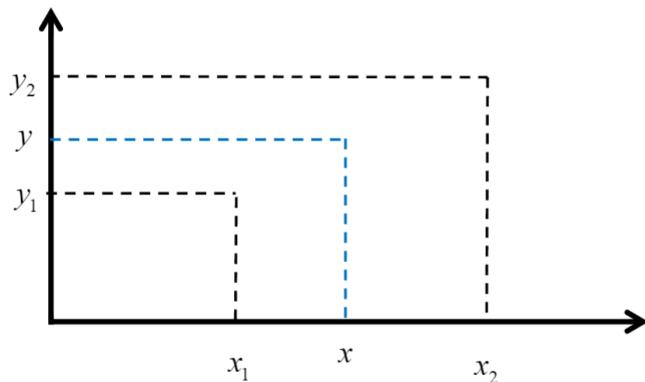
The values of u, h , and s cannot be measured directly, and they are calculated from measurable properties using the relations between properties.

However, those relations give the changes in properties, not the values of properties at specified states.

Therefore, we need to **choose a convenient reference state and assign a value of zero** for a convenient property or properties at that state.

The reference state for water is 0.01°C and for R-134a is -40°C in tables. Some properties may have negative values as a result of the reference state chosen.

Interpolation/extrapolation:



$$y = mx + b$$

$$y = \frac{y_2 - y_1}{x_2 - x_1} (x - x_1) + y_1$$

Using tables examples:

- What is the difference between treating water at 100 °C with the exact value and saturated conditions for the specific volume and internal energy at 5 MPa and 50 MPa?

- ANSWER:

Looking at table A4: pressure well above saturation pressure at 100C \therefore Compressed liquid

Approximating : $y_{CL} \approx y_{sat @T}$

$$v = 0.0010410 \frac{m^3}{kg}$$
$$u = 417.65 \frac{kJ}{kg}$$

Looking at table A7 for compressed liquids:

$$v = 0.00102 m^3.kg$$
$$u = 405.94 \frac{kJ}{kg}$$

\therefore quite small % error

QUESTION 2:

Steam is at 750 kPa and 500°C, what is the entropy (s) of the steam?

Temperature above saturation pressure \rightarrow Superheated vapour

	T	s
600 kPa		
750 kPa		
800 kPa		

Processes and cycles:

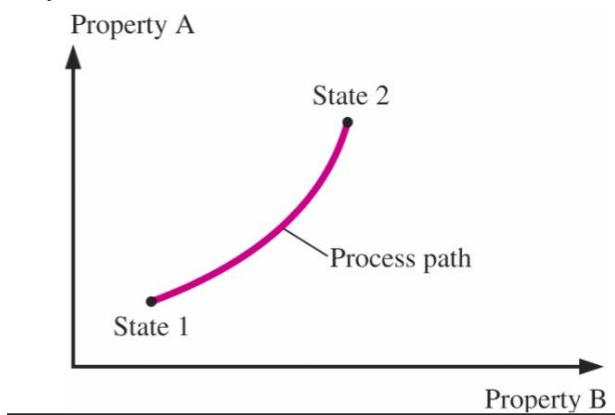
Process:

Any change that a system undergoes from one equilibrium state to another.

Path

The series of states through which a system passes during a process

To describe a process completely, one should specify the initial and final states, as well as the path it follows, and the interactions with the surroundings.



Quasistatic process:

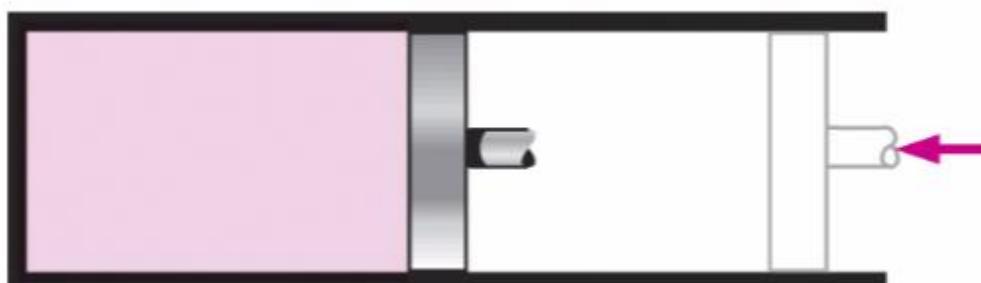
When a process proceeds in such a manner that the system remains infinitesimally close to an equilibrium state at all times.

Nonquasistatic process

When a process proceeds very fast at that any instant the system state is far from equilibrium



(a) Slow compression
(quasi-equilibrium)



(b) Very fast compression
(nonquasi-equilibrium)

Isothermal process:

A process during which the temperature T remains constant.

Isobaric process:

A process during which the pressure P remains constant.

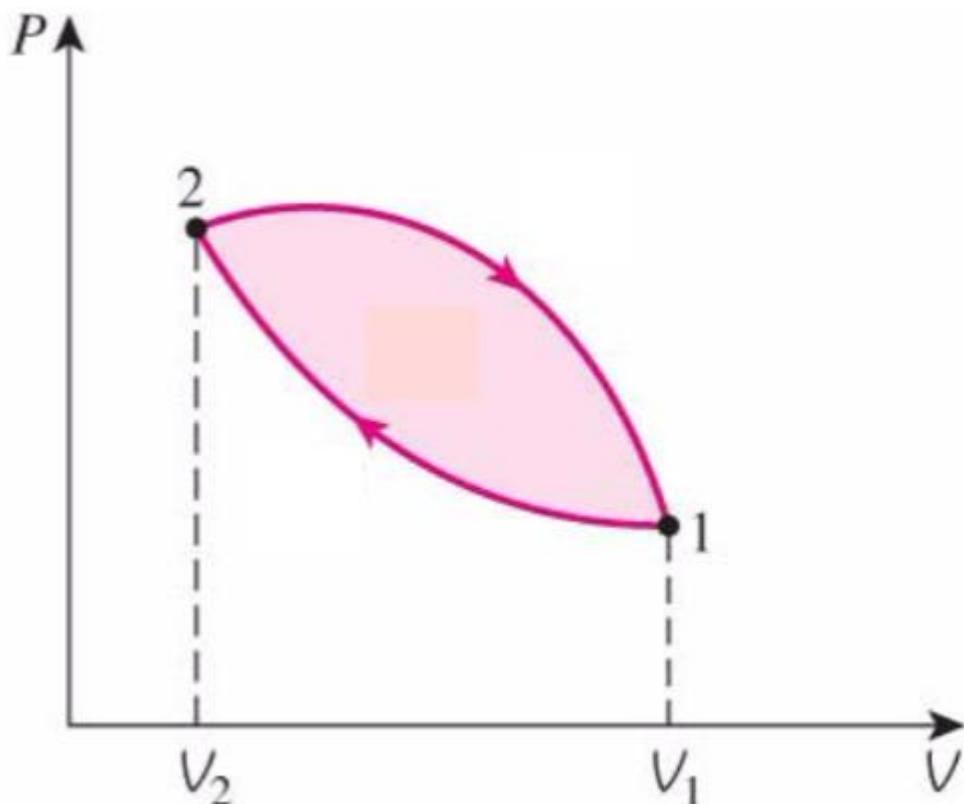
Isochoric (or isometric) process:

A process during which the specific volume v remains constant. Cycle: A process during which the initial and final states are identical.

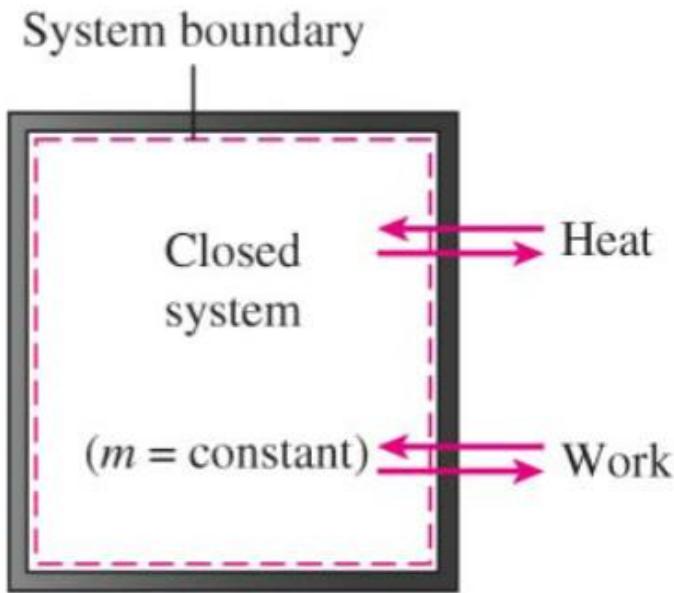
Cycle:

A process during which the initial and final states are identical.

Cycle diagram example:



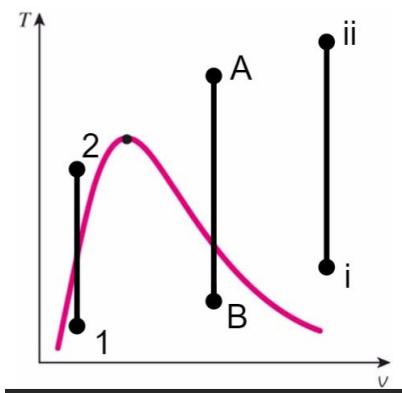
Example Process path: rigid container



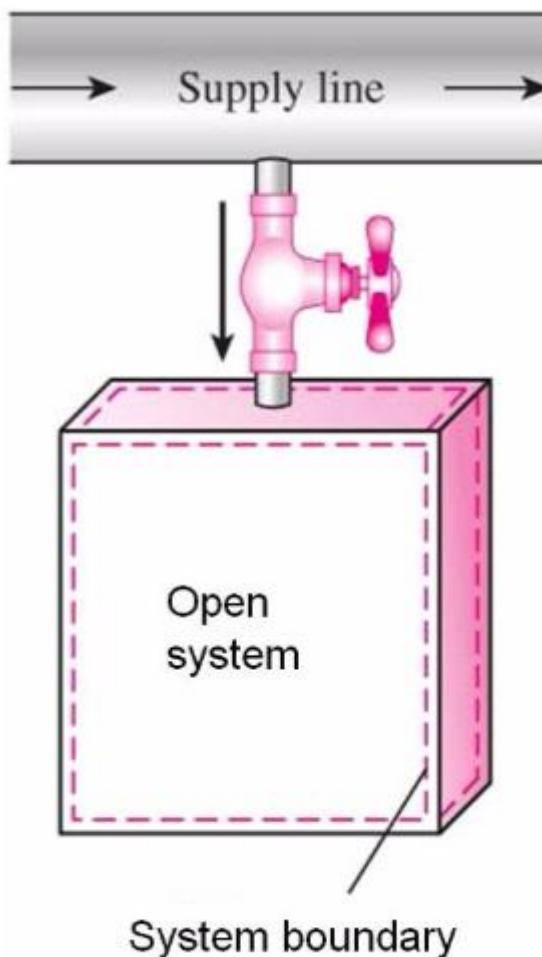
Isochoric process (as rigid container, with mass and volume staying same as control mass)

Therefore pressure and temperature not constant

\therefore vertical path on Tv or Pv diagram



Example 2: open system



As open system, not isochoric → pressure and temperature will be curves in Pv or Tv diagrams

Piston cylinder closed system example:

Closed system:

Volume not constant, mass constant, pressure constant if a free piston- no external pressure)

∴ Horizontal process in Pv diagram

Follows an isobar in Tv diagrams

Example: Process path in rigid container

A rigid container of volume 40 L contains water initially at a pressure of 10 kPa. Energy is added to the water until the pressure at the final state is 50 MPa.

1. Determine the initial and the final temperatures

$$V = 40L = 0.04m^3; P_1 = 10kPa; P_2 = 50MPa$$
$$m_{total} = 7kg \rightarrow v = \frac{0.04}{7}$$

table A5: $T_{sat@10kPa} = 45.81^\circ$
 $v_f = 0.00101; v_g = 14.67$
 $\therefore \text{as } v_g > v > v_f \text{ saturadted liquid.}$
 $\text{therefore } v = v_f + xv_{fg} \rightarrow x = 3.207 \times 10^{-4}$
 $\rightarrow u = u_f + xu_{fg} = 192.51 \frac{kJ}{kg}$

2. ii. Determine the percentage of the volume in the vapor phase at the initial and final state

$$\therefore V_{vap} = m_{total}xv_{vap} = 0.032932 \text{ m}^3$$

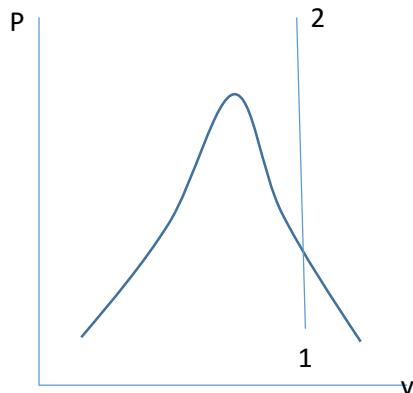
$$\therefore V_{Vap}\% = \frac{0.0329}{0.04} = 82.3\%$$

3. iii. Determine the change in internal energy of the water through the process (in kJ).

$$P_2 = 50MPa; v_2 = v_1 = 0.0057132$$

$$P_{cr} = 22MPa; v_{cr} = 0.003106 \rightarrow \text{superheated vapour}$$

T	v	u
550	.005118	2769.5
580.12	0.0057143	2876.1
600	0.006108	2947.1



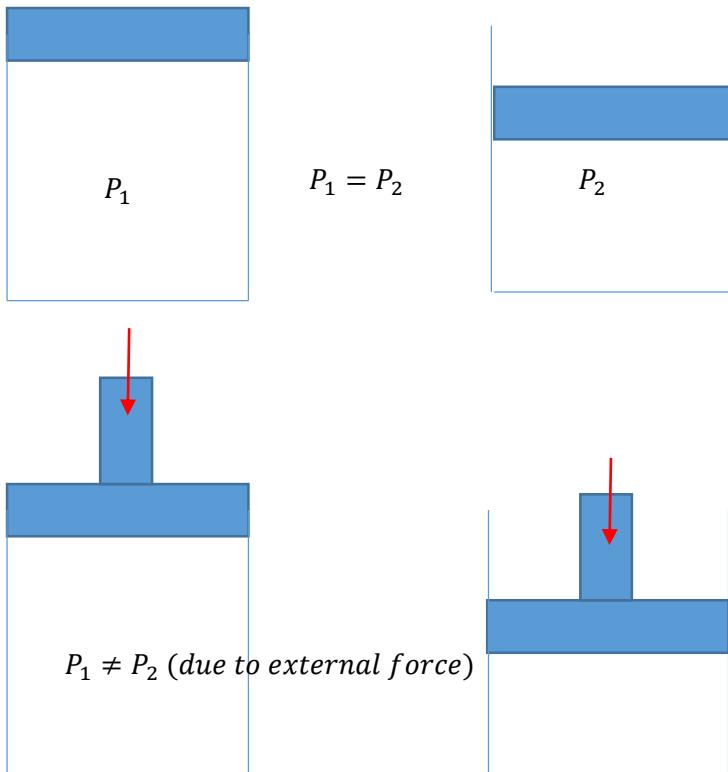
4. iv. Show the process on T-v and P-v diagrams with respect to the initial and final states and the saturation curve.

Example 2:

A rigid container of volume 60 000 cm³ contains water initially at a temperature of 53.1°C. Initially 61% of volume is in the vapor phase and remainder in the liquid phase. Energy is added to the water until the temperature at the final state is 440°C. i. Determine the initial and the final Pressures ii. Determine the percentage of the mass the vapor phase at the initial and final state iii. Determine the change in internal energy of the water through the process (in kJ). iv. Show the process on T-v and P-v diagrams.

v diagrams with respect to the initial and final states and the saturation curve. 5 A very difficult question with lots of interpolation: Final pressure close to 44.8 Mpa...

Process path: Piston cylinders



Compressibility factor:

Compressibility factor Z: A factor that accounts for the deviation of real gases from ideal-gas behavior at a given temperature and pressure.

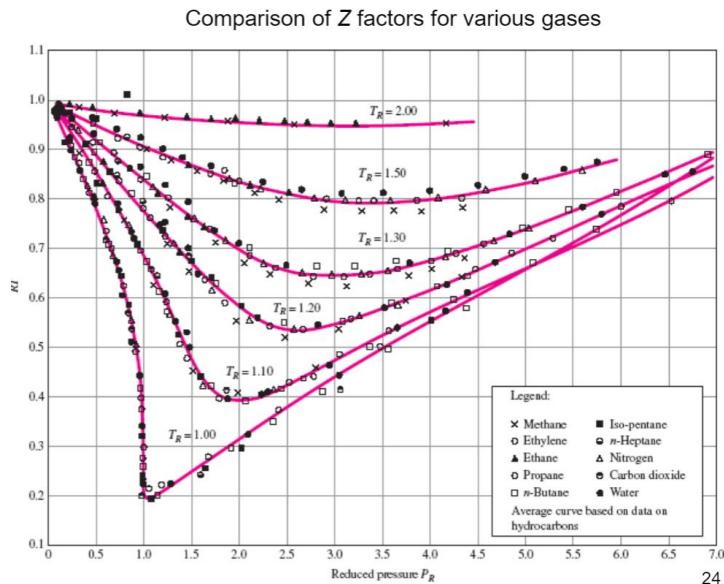
$$Z = \frac{v_{actual}}{v_{ideal}} = \frac{Pv}{RT}$$

for $Z \approx 1$; approximately ideal gass
further Z is from away from 1, less ideal gas like

$$Pv = ZRT$$

NOTE: Z is not a fixed value.

The farther away Z is from unity, the more the gas deviates from ideal-gas behavior. Gases behave as an ideal gas at low densities (i.e., low pressure, high temperature).



Where

Reduced pressure:

$$P_R = \frac{P}{P_{CR}}$$

Reduced temperature

$$T_R = \frac{T}{T_{CR}}$$

Pseudo Reduced specific volume

$$v_R = \frac{v}{\left(\frac{RT_{CR}}{P_{CR}}\right)} = \frac{vP_{CR}}{RT_{CR}}$$